Part 1  General

1.1  Summary

.1 Unless otherwise indicated, follow the standards below when specifying heating, ventilation and air conditioning (HVAC) work. These standards are not intended to restrict or replace professional judgment.

1.2  Références

.1 Refer to section 00 20 00 Instructions to consultants.

.2 The equipment identification must respect the standard established by McGill University, Drawing STD-M-001 see section 23 05 53 Identification des Systèmes Mécaniques et des Dessins.

1.3  Refrigerant Equipment

.1 General

.1 Centrifugal chillers are to be preferred over single stage absorption chillers. Stand-alone rooftop glycol units are to be avoided.

.2 Chillers shall use 134a refrigerant or better ozone friendly for units above 100 tons, shall have proven service and reliability, parts, services, all-inclusive service contract for five (5) years and a five (5) year renewal clause.

.3 Chilled water coils shall have 25% extra capacity of refrigeration with respect to the design load required by the system.

.4 All multicoil DX shall be provided with interlaced type circuiting. Avoid multicoil DX with single compressor.

.5 The professional in charge of design must select the chiller best adapted to the application. In all cases, the designer must demonstrate that the type of chiller has been selected appropriately for the application and based on a total cost of ownership (TCO) analysis including initial investment, annual operation and maintenance costs.

.6 For heat recovery applications of 200 tons and below, modular multi-stage chillers (scroll or screw) are to be preferred over centrifugal chillers because they allow for better modulation and the production of hot water at a higher temperature.

.7 The minimum efficiency of chillers must be as good as or better than ASHRAE 90.1-2010.

.8 Design chillers with Delta-T across the evaporator and condenser of 15°F (8.3°C) to reduce pump energy.

.2 Glycol Re-circulation Systems

.1 Ethylene glycol should preferably be specified, unless the use of propylene glycol is required for the application. However, it is the responsibility of the design engineer to use the appropriate type of glycol for each respective application taking into account applicable laws and regulations.

.2 The following types of glycol should be specified:
1.4 Heating, Ventilation, Air Conditioning Equipment

.1 Design Control for Mechanical Systems

.1 The professionals responsible for system design shall inform the Executive Director Facilities Management and Development in writing whenever budgetary constraints may prevent the performance objectives of this standard from being respected.

.2 Longevity and Performance Criteria for any new system may be introduced if and only if the supplier can provide an obsolescence statement guaranteeing a minimum of ten years serviceability for the system, including the refrigerant gas.

.2 General

.1 Loading docks shall be provided with air curtains or other means to minimize air infiltration and be isolated from the rest of the building. Provide minimum heating at the docks by ducting the building exhaust air to this area. (Use air curtain wall)

.2 Perimeter rooms shall be air-conditioned with sill type grilles and ceiling diffusers (Linear).

.3 All mechanic HVAC equipment shall be housed in a penthouse; rooftop units shall be avoided.

.4 Relative pressures for labs shall be negative with respect to the corridor. Intermediate and high-level radiation labs shall be negative with respect to other labs, and buildings shall be slightly negative to the outside.

.5 If peripheral heating is required, it shall be fed from under the floor. An all-air system with a two supply (hot and cold), one return fan dual duct variable air-column system shall be supplied along exterior walls from under the floor.

.6 The return air shall supply the hot duct 100%; excess air shall supply the cold duct (with outside air).

.7 See section 23 82 23 for HVAC units design.

.3 Mechanical Preheat, Reheat or Cooling Coils

.1 Whenever possible and applicable, the design engineer shall consider using "dual service" type coils (same coil for heating and cooling - depending on the season) filled with glycol and serviced by heat exchangers if required.

.2 All air handling units' coils shall be provided with glycol, unless a variance request has been accepted by the mechanical engineer of McGill Facility Management and Ancillary Services Department.

.3 Integral Face and Bypass (IFB) or Vertical Integral Face and Bypass (VIFB) preheat or reheat coils shall not be used. Glycol heating coils shall be provided instead.

.4 Coils shall be tested in accordance with A.R.I. standard 410.
.5 Cooling coils air velocities shall not exceed 490 ft/min (2.5 m/s).

.6 Coils construction:
   .1 Coil casings shall be 304L stainless steel, 16-gauge minimum.
   .2 Headers shall be copper.
   .3 Water and DX coils tubes shall be 5/8 inch (16 mm) minimum seamless drawn copper, thickness 0.035” minimum.
   .4 Steam coils tubes shall be 1.0 inch (25.4 mm) minimum seamless drawn copper, thickness 0.049” minimum.
   .5 Plate type coil fins shall be plate type aluminum 0.010” thickness, maximum spacing of 10 fins per inch. If helical fins are provided for cooling or heat recovery coils, then stainless steel mist eliminators are required in order to avoid water carryover.

.7 A strainer shall be installed before the control valve on any coil.

.8 All coils shall be equipped with a vacuum breaker at the highest point.

.9 Each hydronic coil shall be provided with isolation valves, balancing valve manometers, thermometers and air vent. The thermometers and manometers shall be installed on the supply and also on the return water sides.

.10 Steam traps shall be installed with recommended slopes for drainage.

.4 Vibration Isolators
   .1 Vibration eliminators shall be provided for all moving equipment called for in all projects. Equipment shall be effectively isolated from the building structure to prevent undue vibration and noise transmission to the building.

.5 Humidification
   .1 Direct steam humidifiers using steam distribution to be preferred over all other alternatives.
   .2 The humidifier dispersion tubes shall be insulated. The tube insulation shall be ceramic or polyvinylidene fluoride (PVDF) – shielded air gap insulation is not accepted.
   .3 If steam humidification is not feasible or financially viable, the design engineer must prefer the following sources in the following order:
      .1 Natural gas
      .2 Electricity.
   .4 Humidifiers shall be ARMSTRONG, DRISTEEM or NORTEC or reusable canister. An approved equivalent manufacturer shall be considered if approved in writing by the representative of McGill’s Facilities Operations Department (FOD).

1.5 Air Distribution

.1 General
   .1 Transformer grilles in doors or transoms are not permitted. Air transfer shall be achieved by duct transfer grilles installed in ceilings, with fire dampers and provision for sound attenuation as required.
   .2 Air intakes shall be located away from vehicle circulation areas and exhausts at roof level, not at street level for car fumes. If this cannot be done, then airflow studies shall be considered with respect to the positioning of air intakes and wind.
Air intake velocity shall be 200 feet per minute or less. All basins in air handling units shall be drained with slopes to drains.

Duct insulation shall be installed on the outside of ductwork. Insulation on the inside of ductwork shall be avoided. Acoustical insulation shall be accomplished with silencers.

Transformer rooms shall have their own exhaust and supply with no heating to be provided.

Mechanical room and washroom exhaust fans shall be stop-start programmable.

Air supply to transformer rooms shall be filtered (using 2" deep - Merv 8 filters, combining standard sizes of 24"x24" and 12"x24" only).

Variable air volume (VAV) ventilation systems to be preferred for any space on campus inasmuch as it responds to occupant and functional requirements.

Spaces that do not require continuous ventilation should be supplied by autonomous systems and shut down at unoccupied times.

Space with high internal heat gains due to the presence of equipment (such as electronic equipment, freezers and refrigerators, mechanical equipment, etc.) must preferably be cooled using chilled water fan coils.

Ductwork shall be as per the latest SMACNA and ASHRAE standards.

If the ductwork needs to be insulated, then the insulation shall be placed on the outer wall of the ventilation ducts (i.e. no insulation allowed in the air tunnel). If required, acoustic silencers shall be provided. In addition, all airstream surfaces should be resistant to mould growth and resist erosion, according to the requirements of ASHRAE Standard 62.1 (ASHRAE 2010a).

The following ductwork shall be insulated to a minimum of R-6:

- All supply air ductwork.
- All return air ductwork located above the ceiling and below the roof.
- All OA ductwork.
- All exhaust and relief air ductwork between the motor-operated damper and penetration of the building exterior.

Flexible conduits:

- Flexible conduits as per NFPA-90, NFPA-90B, ULC.
- Maximum length of 1500 mm (60").
- Acceptable models:
  - Boflex, model types AS and Al.
  - Equipement Trans Continental, model Al-U-Flex.
  - Flexmaster Co Ltd., model Triple lock.

All new ductwork for chemical fume hood exhaust systems shall be 316 stainless steel, low carbon, round, with annealed welds; or fibreglass reinforced plastics depending on the application.

All existing conduits that will be reused need to be sealed:

- Repair all major leaks as per SMACNA recommendations using products that are UL 181 compliant.
.2 Seal the existing conduits from the inside using an automatized system such as Aeroseal or approved equal. The application shall be done by an authorized distributor.

.3 Diffusers

.1 All new and replacement ceiling diffusers shall be selected in order to achieve an efficient air distribution. Shape and colour to be coordinated with the project architect. Provide balancing dampers for each diffuser.

.2 Induction type diffusers shall be used for variable air volume ventilation systems.

.3 Fabric faced terminal diffusion devices are not accepted at McGill, unless a variance request has been accepted by the mechanical engineer of McGill Facility Management and Ancillary Services Department.

.4 Terminal boxes (mixing boxes and VAV boxes)

.1 For each new project, verify that the terminal boxes serving the designated area are still functional and have not exceeded their life expectancy, in which case they would need to be replaced.

.2 All new mixing or variable air volume (VAV) terminal boxes shall be Direct Digital Control (DDC) type and be connected to McGill University’s Building Automation System (BAS). This shall be done even when the existing terminal boxes are pneumatic type.

.3 Each box shall serve a designated zone and be controlled by its dedicated DDC thermostat connected to the BAS system.

.5 Air Filters

.1 For general applications pre-filters shall be MERV 8 2 "(50mm) and final filters shall be MERV 13 - 12” (300mm) rigid box type with header for side-access or front access applications.

.2 Always use standard filter sizes 24”x24” and 12”x24”. Other filters sizes shall be used when it is impossible to use standard size or when the standard size is not available for a given application. When this is the case, notify in writing the maintenance mechanical engineer of McGill University Facilities Management and Ancillary Services.

.3 Filter frames and fasteners

.1 Frames and fasteners shall be from a recognizable filter manufacturer, with a minimum of 5 years in existence. They need to be easily available locally.

.2 The fasteners shall allow for an easy installation or removal of filters, without having to use any tools. In addition, no tool should be required in order to replace any of the filter fasteners.

.3 Preferred fastener type, specific to the installed filter combination: C-78 series by Camfil or Filter clip series by Filtration Lab or approved equivalent.

.6 Dampers and louvres

.1 See section 23 82 23 for a complete description.

.7 Fans

.1 Whenever available, fans shall be direct driven type rather than belt driven.
.8 Access Doors

.1 Access doors shall be provided at locations in the duct systems where access to manual or automatic fire dampers, coils, thermostats or any other apparatus requires inspection.

.2 Doors shall be constructed with an external built-up metal frame for stiffening. Doors on insulated ductwork shall be of double panel construction provided with glass fibreboard filler.

.3 Access doors shall be provided ahead of all fan inlets and on both sides of all coils to allow for cleaning and inspection.

.9 Flexible Connections and Isolating Connections

.1 Flexible connections shall be provided on joints between ducts and air-handling equipment. They shall be 150 mm (6 inch) wide Ventglass type flexible connections. Two 13 mm (1/2 inch) wide heavy flexible braised copper conductors with clamps shall be installed to ground all systems.

.2 Flanges shall be provided on the ducts to make proper connections. Connections shall have a minimum of 100 mm (4 inch) between the flanges and shall be airtight. Similar connections shall be supplied for the joints between dissimilar metals in the ductwork. Flexible conductors shall be used to bridge all flexible connections in ductwork.

1.6 Duct sealing

.1 If required and/or it has been decided that the ductwork needs to be sealed for a particular project: repair any major leaks larger than 12 mm (0.5 in.) in diameter with sealant and fibreglass tape as recommended by SMACNA. The product used must meet the requirements of UL 181.

.2 Seal the existing ducts from the inside using an automated aerosol system.

1.7 Testing, Adjusting, and Balancing

.1 General

.1 Before renovations, all HVAC systems affected by the change shall be checked and airflow readings shall be recorded.

.2 After renovations, all HVAC systems affected by the change shall be checked and balanced.
.3 Commissioning of new HVAC systems shall wait until all construction work is complete and all dust and dust sources have been cleaned up and eliminated.

.4 Air handling units shall not be turned on during construction.

.5 Filters on the return air shall be changed after construction is finished and before final balancing.

.6 Final inspections shall be made to ensure after start-up that filter frames are not damaged, ducts are clean, balancing is correct.

.7 The project contractor has the responsibility to document the test.

.8 The project engineer has the responsibility to conclude on the passage of the test.

.2 Balancing and Testing

.1 All air handling units and their respective distribution networks shall be balanced for air quantities as shown on the engineering drawings.

.2 Trades shall provide all belts and pulleys required for balancing all fan systems.

.3 Trades shall submit all testing and balancing results on 8.5 x 11 inch sheets with cross-referenced drawing showing diffuser location and air quantities removed.

.4 Fan test shall be submitted. The entire document shall be bound in a title binder with hard fibreboard cover. The document shall also be provided in electronic format (PDF is preferred).

1.8 Cooling Towers

.1 Cooling towers and all interior surfaces and components shall be made of 304L stainless steel. The unit’s floor shall be complete with heavy duty 304L stainless steel grating.

.2 The fans shall be direct driven type. If direct driven is not available, provide gearbox driven motor. Belt driven fans shall be avoided.

.3 The motors must be controlled by variable frequency drives – see VFD section 23 09 33.

.4 Cooling towers shall be complete with efficient drift eliminators that reduce drift to a maximum of 0.002% of recirculated water volume for counter-flow towers and 0.005% of recirculated water flow for cross-flow towers.

.5 Provide basin sweeper piping in order to limit debris build up.

.6 Cooling towers shall be complete with 304L stainless steel ladder and railing.

.7 Depending on the cooling tower location and its surroundings there might be a risk that contaminants (dust, pollen, etc.) get into the cooling tower causing clogging and premature stoppage of the unit. Removable stainless steel mesh screens shall be provided at the air intake in order to ensure proper operation.

.8 Cooling tower performance (in US gpm per HP) must follow the latest version of ASHRAE 90.1.

1.9 Heating

.1 General
.1 In all projects and all buildings, the professional in charge of design shall select heating sources in the following order:

.1 Heat recovery from internal gains thanks to a heat recovery chiller or a heat pump or using existing local heat recovery loops;
.2 Central steam distribution via steam to hot water heat exchangers or central hot water distribution;
.3 Geoexchange (i.e., ground sourced heat pump system) energy, especially for new construction or full-building renovation projects;
.4 Natural gas hot water condensing boilers in isolated cases where central steam and hot water distribution networks are not available;
.5 Electric boards or coils when none of the above-mentioned energy sources is available.

.2 Heating networks: heating networks must be designed using low temperature set points for compatibility with existing or upcoming low-temperature heat recovery networks. Design temperature set point must be less than or equal 120°F with a 20°F to 40°F temperature differential.

.3 When using natural gas or electricity for heating purposes, the professional in charge of design must use existing service entrances in order to avoid the addition of energy meters from utility companies.

.2 Building Envelope Heating

.1 Geoexchange to be preferred in new buildings.
.2 For new buildings, the professional in charge of design must prefer the use of low-temperature radiators or the use of hot ventilation air.
.3 The use of steam radiators or convectors is forbidden for building envelope heating and the professional in charge of design must consider converting such equipment to hot water when it is included in the scope of the project.
.4 Use of electric baseboard heaters, electric convectors or electric hot air curtains must only be considered in remote areas far from any hot water or steam distribution. Electric equipment must be equipped with triac relays for applications of less than 1 kW and with SCR controllers for applications greater than or equal 1 kW.
.5 Loading docks, indoor garages, warehouses and other space equipped with garage doors must be equipped with hot water unit heaters. In case there is a risk of freezing, other means shall be considered and approved by Variance Request Form.

.3 Terminal Heating

.1 In any upgrade or installation of an HVAC system comprising terminal heating, the professional in charge of design must prefer the use of low-temperature hot water (120°F and below) with a 20°F to 40°F temperature differential.
.2 The use of electric terminal heating can be acceptable but the professional in charge of design must demonstrate that this is the best option based on a total cost of ownership (TCO) analysis including installation, energy, and maintenance costs.

.4 Heat Recovery and Free Cooling

.1 Consider the use of heat recovery systems when the winter cooling load of a building is greater than 30 tons.
.2 Estimation of a building’s winter cooling load shall include the following internal gains: IT cabinets and server rooms, air exhausts greater than 5,000 CFM, HVAC systems requiring cooling in the winter, all cooling loads 5 tons and over.

.3 Lab air exhausts 5,000 cfm and over must be equipped with heat recovery. Heat recovery systems must prevent cross contamination.

.4 All HVAC systems with air supply 2,360 L/s (5,000 CFM) and over and whose fresh air ratio is greater than 70% must be equipped with a heat recovery system with a total efficiency ratio (sensible and latent) greater than 50%.

.5 Heat recovery is to be preferred over free cooling when possible. In other cases, free cooling must be considered for all systems with a cooling load 10 tons and over.

.6 The professional in charge of design must evaluate whether free cooling or mechanical cooling is the best option for HVAC systems requiring cooling in the winter. The justification of using mechanical cooling must be based on an evaluation of the minimum fresh air requirements based on applicable standards.

.7 Bearing in mind that the chillers designed for summer usage are often oversized for winter applications, the professional in charge of design must consider the use of heat recovery chillers adapted to the minimum cooling load of the building in the winter.

END OF SECTION